

barrel (consequence of the clearance between the driving band of the projectile and the barrel grooves), by static disequilibrium (the projectile centre of mass is outside the axis of symmetry) and dynamic disequilibrium (some mobile parts of the weapon have the centre of mass outside the axis of symmetry).

The differential equation of a constant cross section barrel bending vibration is [2, 3]:

$$EI \frac{\partial^4 y}{\partial x^4} + m \frac{\partial^2 y}{\partial t^2} = F(t) \quad (1)$$

where:

$$I = \frac{\pi}{64} (D^4 - d^4) \quad \text{– moment of inertia for}$$

the barrel cross section;

m – mass of the barrel;

y=y(x) – vertical deflection of barrel axis from its initial position, as a function of x coordinate;

F(t) – acting force, varying in time after a certain law.

2. Description of the equipment

It has been used an optical method to record the bending vibrations of the barrel,

using performant modern technology, meaning high speed camera capable of recording images up to the speed of 1 million frames/second.

Laboratory equipment used for measurements was made available courtesy of the Royal Military Academy Brussels.

The barrel vibrations measurement chain, with accessories, comprises of:

- PHOTRON FASTCAM SAX2 high speed camera;
- computer for saving, editing and post processing data;
- small dimension monitor in the proximity of the camera, for fine manual tuning of the image;
- Drello Muzzle Flash BAL 607D flash detector, used as a trigger;
- high luminosity spotlights, focused on the interest area;
- optionally, when measuring the displacement of two points situated in two different areas of interest, a second high speed camera, connected in slave mode to the first one.

Figure 1: Barrel vibrations measuring chain

Another essential piece of the measurement like resolution or number of frames/second. process is represented by the professional Photron Fast View may also be used to software used: Photron Fast View and play in slow motion the films that have Photron Motion Tools been recorded before.

The first one allows controlling the camera, The second software, Photron Motion Tools, is a powerful instrument to analyse from a computer, in order to change settings

the kinematics of moving objects recorded displacement of the subject area on two with the cameras. For the software to work axes versus time in a graph and an Excel correctly, small surfaces of contrast on the data sheet. studied object are necessary. These surfacesThe weapon was fixed in a ruggedizing can be identified on the object or created by support before firing. The body of the marks. The software has the means to firearm is firmly held both in front and rear calculate the precise plane movement of the sides, by two pairs of screwing sabots. The surface, therefore returning the barrel remains free floating.

Figure 2: The way of fixing the firearm in a ruggedizing support (bench rest)

3. Experimental tests. Data analyzing

The main objectives of the tests were:

- to identify the amplitudes and frequencies of barrel vibrations for an automatic weapon in single fire mode, until vibrations dumping;
- to determine mechanisms dynamics influence over the barrel vibrations for an automatic rifle, during a firing cycle;
- to measure the time needed for full dumping of barrel vibrations after a single fire;
- to determine the influence of remaining barrel bending vibrations over the second bullet fired in burst mode (the phase difference between the first and the second bullet);
- to identify the influence of muzzle devices over barrel bending vibrations in the muzzle cross section (amplitude, frequency, dumping time).

It has been used 3 different muzzle devices for the experimental tests. The muzzle devices used are muzzle brakes (devices 2 and 3 are also compensators).

(1) (2) (3)

Figure 3 Muzzle devices used for firing with the 5.56 assault rifle

First of all it has been recorded the barrel vibrations with and without a muzzle device attached, for single fire mode, before and after the bullet exit time, until the vibrations are close to be dumped. It has been used as muzzle device the muzzle break no 1. The highspeed camera had been set so:

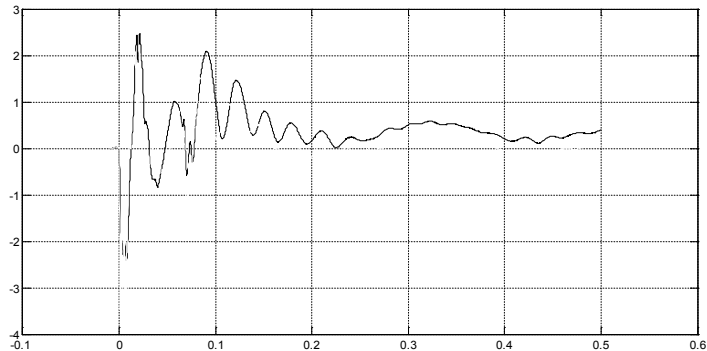
- recording speed: 10000 frames/second;
 - resolution: 256x465;
 - scale: 0.0177 mm/pixel.
- Results are presented in figure 4. The graphics show that:
- the two curves have a comparable profile;
 - the bending oscillation with muzzle break

has an absolute maximum of 2.469mm (positive), compared to 3.699mm (negative) without muzzle break;

- the maxima for the negative and positive displacement of the muzzle are almost equal in absolute values for the vibrations of the barrel with muzzle break, while the negative displacement is 1.6 times higher in

absolute value than the positive one for the vibrations of the barrel without muzzle break, showing that bending vibrations of the barrel in the muzzle section is balanced by the muzzle device;

- 0.2ms after bullet exit, the two curves are very similar, with approx. 0.2mm difference of phase;



(1)

(2)

Figure: 4 (1) Plane trajectory of the muzzle of a 5.56mm assault rifle without muzzle device, single fire
 (2) Comparison between the vertical displacement of the muzzle of a 5.56mm assault rifle without muzzle device (red line) and with muzzle device (blue line), single fire

- the vibrations are not dumped 0.5s after bullet exit time (the end of data acquisition) but extrapolating they will most probably be almost completely dumped in less than 1s after bullet exit time, meaning that they influence very little the shooter in the process of aiming and shooting the next bullet.

Second of all it have been fired 2 bullets in burst mode without muzzle device and with each of the 3 muzzle devices

The settings used for the camera are:

- recording speed: 10000 frames/second;
- resolution: 256x465;
- scale: 0.0177 mm/pixel.

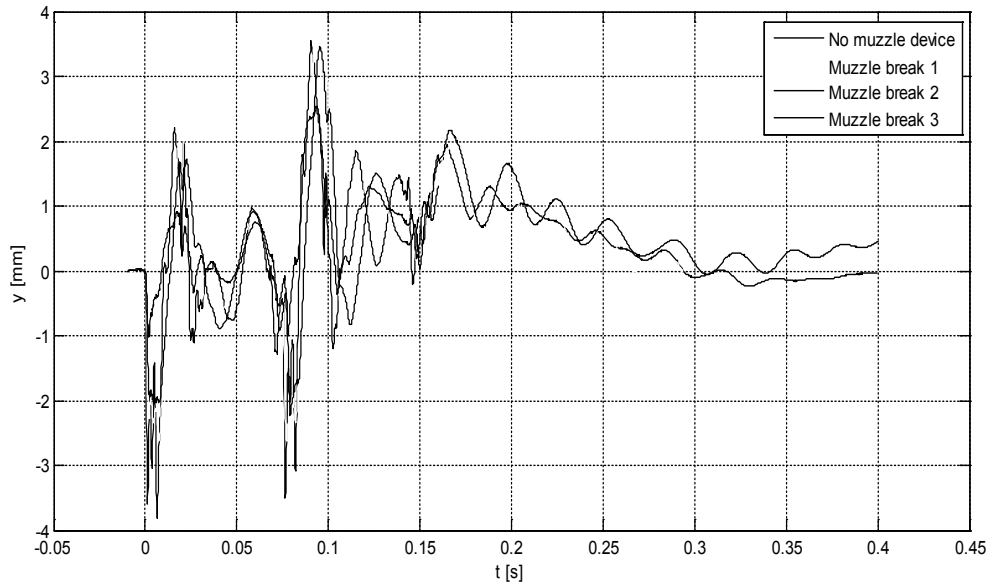
Results are presented in figure 5

(a)

(b)

(c)

(d)



(e)
 Figure: 5 Vertical displacement vs. time for the muzzle of a 5.56mm assault rifle with (a) no muzzle device, (b) muzzle device no 1, (c) muzzle device no 2, (d) muzzle device no 3 2 cartridges burst fire (e) Comparison between the vertical displacement of the muzzle

The bending oscillation with muzzle break no 1, has an absolute maximum of 3.979mm (positive), compared to 3.826mm (negative) without muzzle break. The other two devices generated values between these points. The maxima for the negative and positive displacement of the muzzle are almost equal in absolute values for the vibrations of the barrel with one specified muzzle break, than the one for the vibrations of the barrel without muzzle break, showing that

bending vibrations of the barrel in the muzzle section are balanced by the muzzle device. From the graphics it is obvious that every muzzle device influences the vibratory pattern of the barrel bending vibrations in the muzzle cross section differently. Considering the desired effect on the firing sound attenuation, hidden flame, mitigation or strengthening of recoil, muzzle device configuration will be different.

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References

- [1] *** Rheinmetall Handbook on Weapons, Düsseldorf, 1982
- [2] Roșca Aurel, Automatic weapons The resultant vibration of the barrel. The influence on the firing precision, Military Technical Academy Press House, Bucharest, 2002
- [3] Piersol Allan and Paez Thomas, Harris Shock and vibration Handbook, 6th edition, McGraw Hill Handbooks, 2009